

Multimachine Stabilizer using Sliding Mode Observer-Model Following including CLF for Unmeasurable State Variables

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Abstract

In this paper, the power system stabilizer(PSS) using the sliding mode observer-model following(SMO-MF) with closed-loop feedback (CLF) for single machine system is extended to multimachine system. This multimachine SMO-MF PSS for unmeasurable plant state variables is obtained by combining the sliding mode-model following(SM-MF) including closed-loop feedback(CLF) with the full-order observer(FOO). And the estimated control input for unmeasurable plant state variables is derived by Lyapunov's second method to determine a control input that keeps the system stable. Time domain simulation results for the torque angle and for the angular velocity show that the proposed multimachine SMO-MF PSS including CLF for unmeasurable plant state variables is able to damp out the low frequency oscillation and to achieve asymptotic tracking error between the reference model state and the estimated plant state at different initial conditions and at step input.

I. Introduction

The sliding mode control(or variable structure control) theory [1] has been developed as a controller which offers an effective way of the design of transient stability controllers for power system[2-9]. And the SM-MF[10] including closed-loop feedback(CLF) has been applied to the PSS[12, 13].

These controllers[11-13] are all based on assumption that the complete state is available.

To solve these problems mentioned above the full state feedback, the sliding observer-model following(SMO-MF) for unmeasurable plant state variables has been developed[14].

In this paper, the power system stabilizer(PSS) using the sliding mode observer-model following(SMO-MF) with closed-loop feedback(CLF) for single machine system is extended to multimachine system.

This multimachine SMO-MF is obtained by combining the sliding mode-model following(SM-MF) with closed-loop feedback (CLF) with the full-order observer(FOO).

The estimated control input is derived by Lyapunov's second method to determine a control input that keeps the system stable for unmeasurable plant state variables.

II. Multimachine Generator Model

In this section, we briefly review the three-machine / infinite busbar system.

The block diagram for three-machine/infinite busbar system is represented in Fig. 1[2].

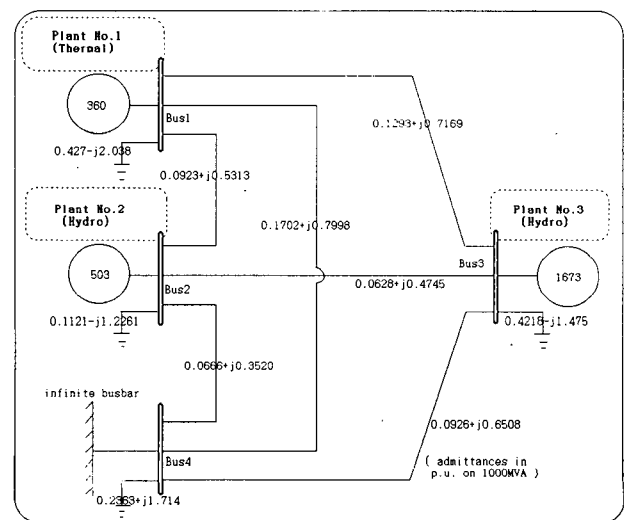


Fig. 1. Three-machine / infinite busbar system.

Manuscript received January 28, 1997; accepted August 11, 1997.

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Fig. 1 is represented by a single equivalent machine with machines 1, 2 and 3, rated 360 MVA(Thermal), 503 MVA(Hydro) and 1673 MVA(Hydro), respectively. And Plant 4 effectively represents an infinite busbar system.

An IEEE type-1 excitation system model for three-machine / infinite busbar system is considered which neglects saturation of the exciter and voltage limits of amplifier output.

The 12th-order state equation for a reference model can be expressed as

$$x_m = [\Delta\delta_{m1}, \Delta\omega_{m1}, \Delta e_{qm1}, \Delta e_{FDm1}, \Delta\delta_{m2}, \Delta\omega_{m2}, \Delta e_{qm2}, \Delta e_{FDm2}, \Delta\delta_{m3}, \Delta\omega_{m3}, \Delta e_{qm3}, \Delta e_{FDm3}]^T \quad (1)$$

where $\Delta\delta_m(t)$ is the torque angle for model, $\Delta\omega_m(t)$ the angular velocity for model, $\Delta e_{qm}(t)$ the q-axis component of voltage behind transient for model and $\Delta e_{FDm}(t)$ the equivalent excitation voltage for model.

The 12th-order state equation for the controlled plant can be expressed as

$$x_p = [\Delta\delta_{p1}, \Delta\omega_{p1}, \Delta e_{qp1}, \Delta e_{FDp1}, \Delta\delta_{p2}, \Delta\omega_{p2}, \Delta e_{qp2}, \Delta e_{FDp2}, \Delta\delta_{p3}, \Delta\omega_{p3}, \Delta e_{qp3}, \Delta e_{FDp3}]^T \quad (2)$$

where $\Delta\delta_p(t)$ is the torque angle for plant, $\Delta\omega_p(t)$ the angular velocity for plant, $\Delta e_{qp}(t)$ the q-axis component of voltage behind transient reactance for plant and $\Delta e_{FDp}(t)$ the equivalent excitation voltage for plant.

III. Multimachine SMO-MF Controller Including CLF for Unmeasurable Plant State Variables

In this section, we present the multimachine SMO-MF including CLF for unmeasurable plant state variables.

The state equation for a reference model can be expressed as

$$\dot{x}_m(t) = A_m \cdot x_m(t) + B_m \cdot u_m(t) \quad (3)$$

where $x_m \in R^n$ is a state vector for model and $u_m \in R^p$ a control input for model. And A_m is a $n \times n$ system matrix for model and B_m a $n \times m$ control vector for model.

The control input of a reference model with r_m can expressed as

$$u_m(t) = -K_m \cdot x_m(t) + r_m(t) \quad (4)$$

where K_m is a $m \times n$ feedback gain for model and can be obtained by pole placement. And $r_m \in R^p$ is a reference input vector for model.

The closed-loop feedback system for a reference model is

$$\dot{x}_m(t) = (A_m - B_m \cdot K_m) \cdot x_m(t) + B_m \cdot r_m(t) \quad (5)$$

$$\text{Let } A_{km} = A_m - B_m \cdot K_m \quad (6)$$

The state equation for the reference model including CLF can be reformed as

$$\dot{x}_m(t) = A_{km} \cdot x_m(t) + B_m \cdot r_m(t) \quad (7)$$

where A_{km} is a $n \times n$ system matrix including CLF for model.

The state equation for the controlled plant with the parameter variations and the output equation can be formed as

$$\begin{aligned} \dot{x}_p(t) &= (A_p + \Delta A_p) \cdot x_p(t) + (B_p + \Delta B_p) \cdot u_p(t) \\ &= \tilde{A}_p \cdot x_p(t) + \tilde{B}_p \cdot u_p(t) \end{aligned} \quad (8)$$

$$y_p(t) = C_p \cdot x_p(t) \quad (9)$$

where $x_p \in R^n$ is a state vector for plant, $u_p \in R^m$ a control input for plant and $y_p \in R^l$ an available measured output for plant.

$\tilde{A}_p = A_p + \Delta A_p$ is a $n \times n$ system matrix with the parameter variations for plant.

$\tilde{B}_p = B_p + \Delta B_p$ is a $n \times m$ control matrix with the parameter variations for plant.

C_p is the $m \times n$ output matrix for plant.

The following full-order observer equation of the controlled plant for unmeasurable state variables can be expressed as

$$\begin{aligned} \dot{\hat{x}}_p(t) &= \tilde{A}_p \cdot \hat{x}_p(t) + \tilde{B}_p \cdot u_p(t) + L_p \cdot (y_p(t) - C_p \cdot \hat{x}_p(t)) \\ &= (\tilde{A}_p - L_p \cdot C_p) \cdot \hat{x}_p(t) + \tilde{B}_p \cdot u_p(t) + L_p \cdot y_p(t) \end{aligned} \quad (10)$$

where $\hat{x}_p \in R^n$ is the estimated state for plant.

$$L_p = P_p \cdot C_p^T \cdot R_p^{-1} \quad (11)$$

is the $n \times m$ output injection matrix for plant.

P_p is the symmetric positive definite solution of

$$\tilde{A}_p \cdot P_p + P_p \cdot \tilde{A}_p^T - P_p \cdot C_p^T \cdot R_p^{-1} \cdot C_p \cdot P_p + Q_p = 0 \quad (12)$$

Q_p and R_p are positive definite matrices chosen by the designer.

From eq. (10), the following assumptions are made:

(\tilde{A}_p, \tilde{B}_p) is controllable and (\tilde{A}_p, C_p) is observable.

The input control vector with a feedback gain for unmeasurable plant state variables is expressed

$$\begin{aligned} u_p(t) &= u_{CLF}(t) + u_{SMO}(t) \\ &= -K_p \cdot \hat{x}_p(t) + u_{SMO}(t) \end{aligned} \quad (13)$$

where $u_{CLF}(t)$ is the closed-loop feedback control input and $u_{SMO}(t)$ the sliding mode observer control input. And K_p is a $m \times n$ feedback gain for plant and can be obtained by pole

placement.

Substituting eq. (13) into eq. (10), for unmeasurable state, the following full-order observer equation of the controlled plant including CLF can be expressed as

$$\begin{aligned} \dot{\hat{x}}_p(t) &= (\bar{A}_p - L_p \cdot C_p) \cdot \hat{x}_p(t) + \bar{B}_p \cdot u_p(t) + L_p \cdot y_p(t) \\ &= (\bar{A}_p - L_p \cdot C_p) \cdot \hat{x}_p(t) + \bar{B}_p \cdot (-K_p \cdot \hat{x}_p(t) + u_{SMO}(t)) \\ &\quad + L_p \cdot y_p(t) \tag{14} \\ &= (\bar{A}_p - \bar{B}_p \cdot K_p - L_p \cdot C_p) \cdot \hat{x}_p(t) + \bar{B}_p \cdot u_{SMO}(t) + L_p \cdot y_p(t) \\ &= (\bar{A}_{kp} - L_p \cdot C_p) \cdot \hat{x}_p(t) + \bar{B}_p \cdot u_{SMO}(t) + L_p \cdot y_p(t) \end{aligned}$$

where $\bar{A}_{kp} = (\bar{A}_p - \bar{B}_p \cdot K_p)$ is a $n \times n$ system matrix with the parameter variations including CLF for plant.

The error vector and the differential error vector to minimize the error in the difference between the state $x_m(t)$ of the reference model and the estimated state \hat{x}_p of the controlled plant for unmeasurable state variables can be expressed as

$$e(t) = x_m(t) - \hat{x}_p(t) \tag{15}$$

$$\dot{e}(t) = \dot{x}_m(t) - \dot{\hat{x}}_p(t) \tag{16}$$

By substituting eq. (7) and eq. (14) into eq. (16), we have

$$\begin{aligned} \dot{e}(t) &= \dot{x}_m(t) - \dot{\hat{x}}_p(t) \\ &= [A_{km} \cdot x_m(t) + B_m \cdot r_m(t)] - [(\bar{A}_{kp} - L_p \cdot C_p) \cdot \hat{x}_p(t) \\ &\quad + \bar{B}_p \cdot u_{SMO}(t) + L_p \cdot y_p(t)] \tag{17} \end{aligned}$$

$$x_m(t) = e(t) + \hat{x}_p(t) \tag{18}$$

By substituting eq. (18) into eq. (17), we have

$$\begin{aligned} \dot{e}(t) &= A_{km} \cdot x_m(t) + B_m \cdot r_m(t) - (\bar{A}_{kp} - L_p \cdot C_p) \cdot \hat{x}_p(t) \\ &\quad - \bar{B}_p \cdot u_{SMO}(t) - L_p \cdot y_p(t) \\ &= A_{km} \cdot (e(t) + \hat{x}_p(t)) + B_m \cdot r_m(t) - (\bar{A}_{kp} - L_p \cdot C_p) \cdot \hat{x}_p(t) \\ &\quad - \bar{B}_p \cdot u_{SMO}(t) - L_p \cdot y_p(t) \\ &= A_{km} \cdot e(t) - (\bar{A}_{kp} - L_p \cdot C_p - A_{km}) \cdot \hat{x}_p(t) + B_m \cdot r_m(t) \\ &\quad - \bar{B}_p \cdot u_{SMO}(t) - L_p \cdot y_p(t) \tag{19} \end{aligned}$$

Suppose the sliding mode exists on all hyperplanes.

The sliding surface vector and the differential sliding surface vector can be expressed as

$$s(e(t)) = G^T \cdot e(t) \tag{20}$$

$$\dot{s}(e(t)) = G^T \cdot \dot{e}(t) \tag{21}$$

where G^T is the sliding surface gain. And the calculation algorithm of G^T for eq. (20) has been found in reference[2, 10, 12].

We introduce the Lyapunov's function to determine a control law that keeps the system on $s(e(t)) \Rightarrow 0$

$$V(e(t)) = s^2(e(t))/2 \tag{22}$$

The time derivative of $V(e(t))$ is given by

$$\dot{V}(e(t)) = s(e(t)) \cdot \dot{s}(e(t)) \tag{23}$$

$$= G^T \cdot e(t) \cdot G^T \cdot \dot{e}(t) \tag{24}$$

By substituting eq. (19) into eq. (24), we have

$$\begin{aligned} \dot{V}(e(t)) &= G^T \cdot e(t) \cdot G^T \cdot [A_{km} \cdot e(t) - (\bar{A}_{kp} - L_p \cdot C_p - A_{km}) \\ &\quad \cdot \hat{x}_p(t) + B_m \cdot r_m(t) - \bar{B}_p \cdot u_{SMO}(t) - L_p \cdot y_p(t)] \\ &= G^T \cdot e(t) \cdot [G^T \cdot A_{km} \cdot e(t) \\ &\quad - G^T \cdot (\bar{A}_{kp} - L_p \cdot C_p - A_{km}) \cdot \hat{x}_p(t) + G^T \cdot B_m \cdot r_m(t) \\ &\quad - G^T \cdot \bar{B}_p \cdot u_{SMO}(t) - G^T \cdot L_p \cdot y_p(t)] \\ &\leq 0 \tag{25} \end{aligned}$$

From eq. (25), the control input vector with switching for the controlled plant can be represented by

$$\begin{aligned} u_{SMO}^+(t) &\geq (G^T \cdot \bar{B}_p)^{-1} \cdot [G^T \cdot A_{km} \cdot e(t) \\ &\quad - G^T \cdot (\bar{A}_{kp} - L_p \cdot C_p - A_{km}) \cdot \hat{x}_p(t) \\ &\quad + G^T \cdot B_m \cdot r_m(t) - G^T \cdot L_p \cdot y_p(t)] \\ &\quad \text{for } G^T \cdot e(t) > 0 \tag{26} \end{aligned}$$

$$\begin{aligned} u_{SMO}^-(t) &\leq (G^T \cdot \bar{B}_p)^{-1} \cdot [G^T \cdot A_{km} \cdot e(t) \\ &\quad - G^T \cdot (\bar{A}_{kp} - L_p \cdot C_p - A_{km}) \cdot \hat{x}_p(t) \\ &\quad + G^T \cdot B_m \cdot r_m(t) - G^T \cdot L_p \cdot y_p(t)] \\ &\quad \text{for } G^T \cdot e(t) < 0 \tag{27} \end{aligned}$$

From eq. (26) and eq. (27), the control input vector with sign function for the controlled plant can be reformed

$$\begin{aligned} u_{SMO}^{sign}(t) &= [SE_{gain} \cdot e(t) + SP_{gain} \cdot \hat{x}_p(t) + SU_{gain} \cdot r_m(t) \\ &\quad + SO_{gain} \cdot y_p(t)] \cdot \mu \cdot \text{sign}(s(e(t))) \tag{28} \end{aligned}$$

where μ is the bias gain.

$$SE_{gain} := (G^T \cdot \bar{B}_p)^{-1} \cdot G^T \cdot A_{km} \tag{29}$$

is a sliding equal error feedback gain.

$$SP_{gain} := -(G^T \cdot \bar{B}_p)^{-1} \cdot G^T \cdot (\bar{A}_{kp} - A_{km} - L_p \cdot C_p) \tag{30}$$

is a sliding equal estimated plant feedback gain.

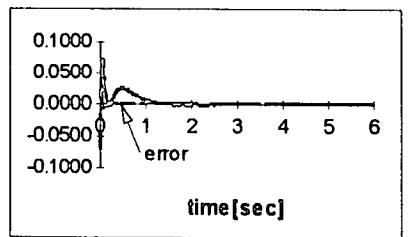
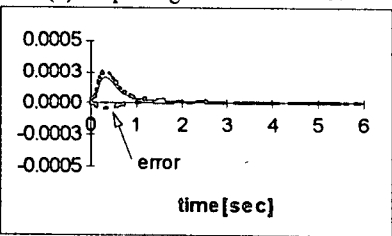
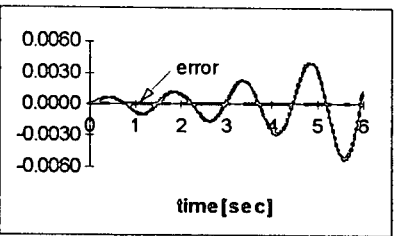
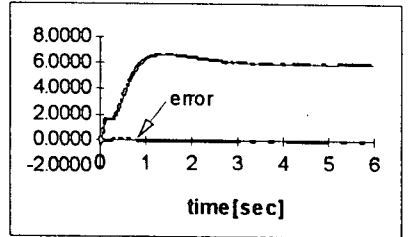
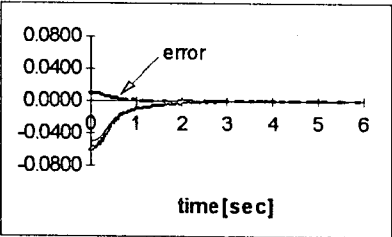
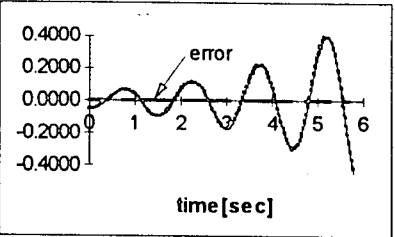
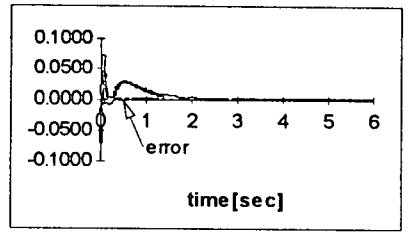
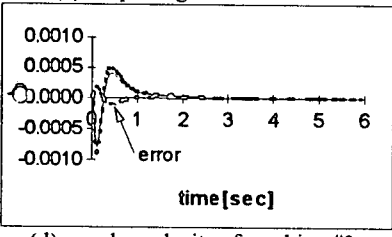
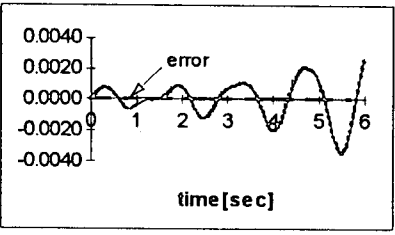
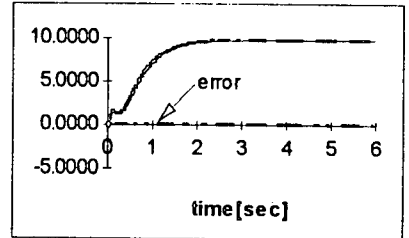
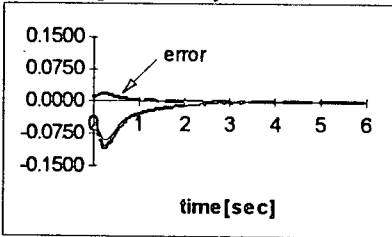
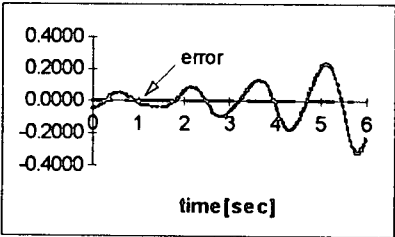
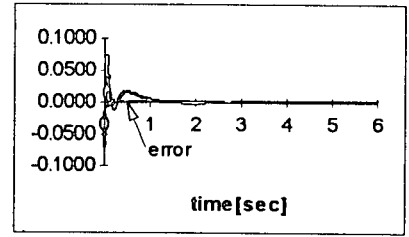
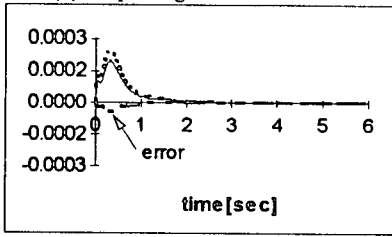
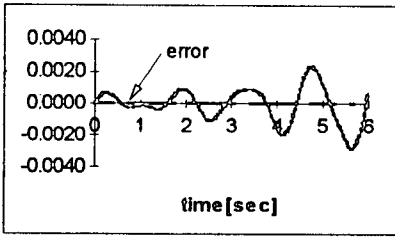
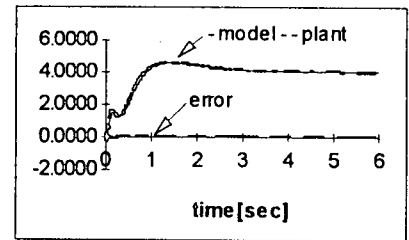
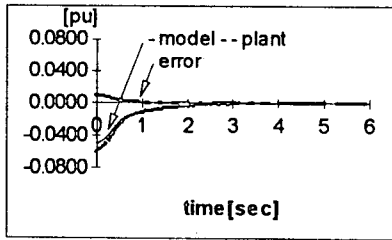
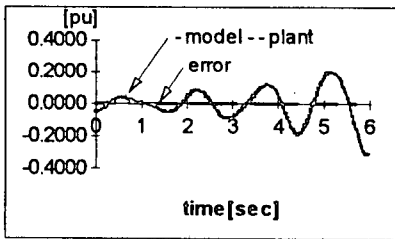
$$SU_{gain} := (G^T \cdot \bar{B}_p)^{-1} \cdot G^T \cdot B_m \tag{31}$$

is a sliding equal input gain.

$$SO_{gain} := -(G^T \cdot \bar{B}_p)^{-1} \cdot G^T \cdot L_p \tag{32}$$

is a sliding equal measured output gain.

The detailed block diagram of the proposed SMO-MF including CLF for unmeasurable plant state variables in Fig. 2 is shown as



(f) angular velocity of machine #3.

Fig. 3. The SMO-MF waveforms without CLF at
 $x_m(0)=[-0.05\ 0\ 0\ -0.05\ 0\ 0\ -0.05\ 0\ 0\ 0]$
 $\hat{x}_p(0)=[-0.05\ 0\ 0\ -0.05\ 0\ 0\ -0.05\ 0\ 0\ 0]$

Fig. 4. The proposed SMO-MF waveforms with CLF at
 $x_m(0)=[-0.05\ 0\ 0\ -0.05\ 0\ 0\ -0.05\ 0\ 0\ 0]$
 $\hat{x}_p(0)=[-0.06\ 0\ 0\ -0.06\ 0\ 0\ -0.06\ 0\ 0\ 0]$

Fig. 5. The proposed SMO-MF step-input waveforms with CLF at
 $x_m(0)=[0\ -0.05\ 0\ 0\ 0\ -0.05\ 0\ 0\ -0.05\ 0\ 0]$
 $\hat{x}_p(0)=[0\ -0.07\ 0\ 0\ 0\ -0.07\ 0\ 0\ -0.07\ 0\ 0]$

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Appendix

A. 1 : Machine data and terminal condition data

Table 1. Machine data,

Plant	X_d	X_d'	X_q	X_q'	T_{do}	T_{do}'	T_{ao}	H
1	0.32	1.68	1.66	0.19	4.0	1.9	1.9	2.31
2	0.33	0.88	0.53	0.22	8.0	0.02	0.04	3.4
3	0.20	1.02	0.57	0.14	7.76	0.04	0.09	4.63
4	0.001	-	-	-	-	-	-	60

Table 2. Terminal condition data.

Plant	$P_o(MVA)$	$Q_o(MVA)$	$V_o(p.u.)$	δ [degree]
1	26.5	37.0	1.03	10.0
2	518.0	-31.5	1.025	32.52
3	1582.0	-49.9	1.03	45.95
4	410.0	49.1	1.06	20.69

A. 2 : System matrix(A_m) and control matrix(B_m)

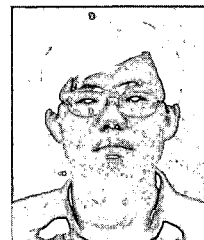
$$A_{m11} = \begin{bmatrix} 0 & 377 & 0 & 0 & 0 & 0 \\ -0.147 & -0.039 & -0.013 & 0 & 0.022 & 0.004 \\ -0.266 & -0.393 & -0.922 & 1 & -0.087 & 0.754 \\ -30.10 & -309.14 & -60.943 & -20 & 24.599 & -91.99 \\ 0 & 0 & 0 & 0 & 0 & 377 \\ 0.004 & -0.034 & -0.087 & 0 & -0.149 & 0.032 \end{bmatrix}$$

$$A_{m12} = \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0.046 & 0.02 & 0.003 & 0 \\ 0.024 & 0 & -0.025 & 1.131 & 0.072 & 0 \\ -3.501 & 0 & 62.051 & -1675 & -10.194 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -0.008 & 0 & 0.097 & -0.028 & 0 & 0 \end{bmatrix}$$

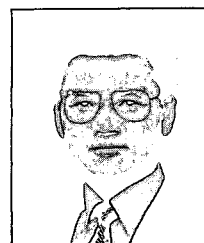
$$A_{m21} = \begin{bmatrix} 0.121 & 1.131 & 0.021 & 0 & -1.6 & -1.885 \\ -18.48 & -64.47 & -12.55 & 0 & 106.09 & -516.11 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0.001 & -0.017 & -0.003 & 0 & 0.017 & -0.01 \\ 0.083 & 0 & -0.002 & 0 & 0.22 & 0 \\ -10.1 & -33.93 & -6.78 & 0 & 1.7 & -46.37 \end{bmatrix}$$

$$A_{m22} = \begin{bmatrix} -0.21 & 1 & 0.46 & 0.754 & 0.06 & 0 \\ -21.67 & -20 & 16.99 & -171.91 & -11.41 & 0 \\ 0 & 0 & 0 & 377 & 0 & 0 \\ 0 & 0 & -0.056 & -0.017 & -0.009 & 0 \\ 0.011 & 0 & -1.2 & -1.131 & -0.19 & 1 \\ -2.1 & 0 & 70.1 & -893.49 & -54.4 & -20 \end{bmatrix}$$

$$B_m = \begin{bmatrix} 0 & 0 & 0 & 800 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 900 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 1000 & 0 \end{bmatrix}^T$$



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