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Study on the single bubble growth at saturated pool boiling

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Key Words :Bubble growth(
Initial growth region(), Pool nucleate boiling(
), Thermal growth region(
)), Microscale heater(
),
), Thermal growth region(
)

Abstract

Nucleate boiling experiments with constant wall temperature of heating surface were performed using R113 for almost saturated pool boiling conditions. A microscale heater array and Wheatstone bridge circuits were used to maintain a constant wall temperature condition and to measure the heat flow rate with high temporal and spatial resolutions. Bubble images during the bubble growth were taken as 5000 frames a sec using a high-speed CCD camera synchronized with the heat flow rate measurements. The geometry of the bubble during growth time could be obtained from the captured bubble images. The bubble growth behavior was analyzed using the new dimensionless parameters for each growth regions to permit comparisons with previous results at the same scale. We found that the new dimensionless parameters can describe the whole growth region as initial and later respectively. The comparisons showed good agreement in the initial and thermal growth regions. The required heat flow rate for the volume change of the observed bubble was estimated to be larger than the instantaneous heat flow rate measured at the wall. Heat, which is different from the instantaneous heat supplied through the heating wall, can be estimated as being transferred through the interface between bubble and liquid even with saturated pool conditions. This phenomenon under a saturated pool condition needs to be analyzed and the data from this study can supply the good experimental data with the precise boundary condition (constant wall temperature).

		\dot{q} +		[-]
		r		[<i>mm</i>]
A, B, C, D, E	[<i>mm</i>]	R		[<i>mm</i>]
Cp,	$\begin{bmatrix} J \\ kgK \end{bmatrix}$	R_{c}		[<i>mm</i>]
h _{fa}	$\begin{bmatrix} J/kg \end{bmatrix}$	R_d		[<i>mm</i>]
k_1	[W/mK]	R _{ea} 가		[<i>mm</i>]
Ja Jakob	[-]	R_{ref}		[<i>mm</i>]
<i>m</i>	[kg / sec]	R^+		[-]
$\Delta P \qquad (P_v - P_{\infty})$	[Pa]	t		[sec]
P_{v}	[<i>Pa</i>]	t		[sec]
P_{∞} ()	[<i>Pa</i>]	t		[sec]
\dot{q}	[W]	ref		[500]
\dot{q}_{c}	[W]	t T		[-]
<i>a</i> .	[W]	T		[K]
A latent		T_{c}		[K]
<i>Y</i> conduction	["]	T_{h}		[<i>K</i>]
t		T_{sat}		[<i>K</i>]
E-mail : jebikim@postech.ac.kr		T_{wall} 7		[K]
TEL : (054)279-5807 FAX : (054)279-3199		ΔT	$(T_{wall} - T_{sat})$	[K]
		V		$[m^3]$

V_{U}		[m ³]	. Mikic	Rohsenow ⁽¹¹⁾	가
V_L		[m ³]			
Greek Letters		2 .		,	
α		$[m^2/s]$	Mikic et al (12)		
$ ho_l$		$[kg/m^3]$	가 가		
$\rho_v = \sigma$		$\begin{bmatrix} kg/m^2 \end{bmatrix}$ $\begin{bmatrix} N/m \end{bmatrix}$		t+<<1	
0		[11, 1, 11, 1]	[+>>] Dobinson	1/2	
	1.		. Koomson	Judu	

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- 7 . (Han Griffith ⁽²⁾, Cole Shulman ⁽³⁾) 7
- ア・ micro
- Rule Kim⁽⁴⁾ Lee et al⁽⁵⁾.
- al ⁽⁶⁾ Rule Kim ⁽⁴⁾ , ⁽⁷⁾ 7
- Rayleigh ⁽⁸⁾
- . Plesset Rayleigh-Plesset
- 가
- . Plesset Zwick ⁽⁹⁾ Forster Zuber ⁽¹⁰⁾ Rayleigh
 - 가 가

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- (Surface-tension controlled region), 가 フト
- 7
 (Inertia controlled region)
 - . (Interface Cooling Effect) 7
 - Plesset Zwick ⁽⁹⁾ Zuber ⁽¹³⁾ Mikic et al ⁽¹¹⁾
 - 가 Han Griffith ⁽²⁾ N-Shulman⁽³⁾ Pentane Cole Hooper (14) Sernas Hooper Abdelmessih (15) Lee et al $^{(16)}$ 1000 frame CCD
 - R11
 - ·
 - R113 (: 47.6 °C) . 가
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$$7 + ...$$

$$V = V_U + V_L = \frac{2}{3} \pi B^2 A + \pi B^2 \left[D - \frac{D^3}{3E^2} \right] = \frac{4}{3} \pi R_{eq} \quad (1)$$

$$7 + ...$$

$$\dot{q} = \dot{m}h_{fg} = 4\pi\rho_v h_{fg} R^2 \frac{dR}{dt}$$
(2)
(2)

$$R = \left(R_{ref}^{3} + \frac{3}{4\pi B^{2} [3 - 2B] \rho_{\nu} h_{fg}} \int_{t_{ref}}^{t} \dot{q} dt \right)^{1/3}$$
(3)

Fig. 2 Geometry of a truncated sphere

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, driving potential . 2/3 Mikic et al ⁽¹¹⁾

$$v_c = \frac{R_c}{t_c} = \sqrt{\frac{2}{3} \frac{\Delta P}{\rho_l}}$$
(4)

Fig. 1 Schematics of the experimental apparatus

$$\frac{q_{latent}}{\dot{q}_{conduction}} = \frac{\rho_{\nu}h_{fg}}{k_{l}4\pi R^{2}} \frac{4}{3}\pi R^{3}}{k_{l}R^{2}} = \frac{1}{3} \frac{\rho_{\nu}h_{fg}R^{3}_{c}}{k_{l}R^{2}_{c}} \frac{R^{+3}}{R^{+2}} = t_{c} \frac{R^{+}}{\frac{\partial T^{+}}{\partial r^{+}}} = t_{c} \frac{R^{+}}{\frac{\partial$$

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Ja $(\Delta T = T_{wall} - T_{sat})$ $(\rho_l C_{P_l} \Delta T) / (\rho_v h_{fg})$. (4) (5) ,

$$R_{c} = \sqrt{\frac{27}{2}} Ja \,\alpha \,\sqrt{\frac{\rho_{l}}{\Delta P}}, t_{c} = \frac{9}{2} Ja \,\alpha \,\frac{\rho_{l}}{\Delta P} \tag{6}$$

$$R^+ = \frac{R}{R_c}, \quad t^+ = \frac{t}{t_c} \tag{7}$$

Mikic et al Rayleigh-Plesset

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$$7$$

$$\Delta P = P_v - P_{\infty} = \rho_l R \frac{d^2 R}{dt^2} + \frac{3}{2} \rho_l \left(\frac{dR}{dt}\right)^2 + \frac{2\sigma}{R} \qquad (8)$$

가

$$v_{c} = \frac{R_{c}}{t_{c}} = \sqrt{\frac{2}{3}} \frac{\Delta P}{\rho_{l}} = \sqrt{\frac{2}{3}} \frac{\rho_{v} h_{fg} \Delta T}{\rho_{l} T_{s}}$$
(9)
Plesset Zwick ⁽⁹⁾

Clausius-Clapeyron

$$R_{c} = \frac{\frac{12}{\pi}Ja^{2}\alpha}{\sqrt{\frac{\pi}{7}\frac{\rho_{v}h_{fg}\Delta T}{\rho_{l}T_{s}}}}, \quad t_{c} = \frac{\frac{12}{\pi}Ja^{2}\alpha}{\frac{\pi}{7}\frac{\rho_{v}h_{fg}\Delta T}{\rho_{l}T_{s}}}$$
(10)

Mikic et al (11)

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(6) Clausius-Clapeyron

(10)

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$$\Delta P = \frac{2\sigma}{R_d} \tag{11}$$

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$$R_{c} = \frac{\sqrt{27}}{2} Ja \,\alpha \,\sqrt{\frac{\rho_{l}R_{d}}{\sigma}}, t_{c} = \frac{9}{4} Ja \,\alpha \,\frac{\rho_{l}R_{d}}{\sigma}$$
(12)

$$\dot{q}_c = 4\pi \rho_v h_{fg} R_c^2 \frac{R_c}{t_c}$$
(13)
(12) ,

$$\dot{q}_c = 54 \frac{1}{\sqrt{3}} \pi \rho_v h_{fg} J a^2 \alpha^2 \sqrt{\frac{\rho_l R_d}{\sigma}}, \quad \dot{q}^+ = \frac{\dot{q}}{\dot{q}_c}$$
(14)

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$$\Delta t = t_c \Delta t^+ = \frac{9}{4} Ja \,\alpha \,\frac{\rho_l R_d}{\sigma} \times \frac{1}{16} \tag{15}$$

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3.4×10⁻⁶ (0.294

(8) Rayleigh-Plesset 3 . Robinson Judd ⁽¹⁾



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