Optimal Design of Ferromagnetic Pole Pieces for Transmission Torque Ripple Reduction in a Magnetic-Geared Machine

Sung-Jin Kim*, Eui-Jong Park* and Yong-Jae Kim†

Abstract – This paper derives an effective shape of the ferromagnetic pole pieces (low-speed rotor) for the reduction of transmission torque ripple in a magnetic-geared machine based on a Box-Behnken design (BBD). In particular, using a non-linear finite element method (FEM) based on 2-D numerical analysis, we conduct a numerical investigation and analysis between independent variables (selected by the BBD) and reaction variables. In addition, we derive a regression equation for reaction variables according to the independent variables by using multiple regression analysis and analysis of variance (ANOVA). We assess the validity of the optimized design by comparing characteristics of the optimized model derived from a response surface analysis and an initial model.

Keywords: Magnetic-geared machine(MGM), Transmission torque ripple, Box-Behnken design (BBD), Finite element method(FEM), Analysis of variance(ANOVA), Response surface methodology (RSM)

1. Introduction

The magnetic gear (MG) is a non-contact machine for torque transmission, acceleration, and deceleration. MGs have advantages such as no mechanical loss and maintenance-free operation [1-6]. Furthermore, MGs have inherent protection characteristics. A variety of MGs have been proposed. On the other hand, a novel structure for a magnetic-geared machine (MGM) was proposed by Atallah et al [1]. This MGM has a high- and low-speed rotor, and a stator. The stator uses coils instead of permanent magnets (PMs), and the low-speed rotor is sandwiched between the high-speed rotor and the stator. The high-speed rotor is driven by the electromotive force generated by the coils, and the low-speed rotor rotates in accordance with the gear ratio [4]. However, these MGMs exhibit high transmission torque ripple, which leads to noise and vibrations in the machine. Therefore, this paper proposes an effective shape of the ferromagnetic pole pieces (low-speed rotor) to reduce the transmission torque ripple of the MGM. The shape is based on a Box-Behnken design (BBD). Using a non-linear finite element method (FEM) based on 2-D numerical analysis, an analysis is conducted between independent variables (selected by the BBD) and reaction variables. In addition, a regression equation of the reaction variable according to the independent variable, using multiple regression analysis and analysis of variance (ANOVA), is derived. The validity of the optimized design is assessed by comparing the characteristics of an optimized model derived from response surface analysis with those of an initial model.

2. Structure and Operational Principle of the Machine

2.1 Analysis model

Fig. 1 shows the MGM in this paper. The high-speed rotor has a yoke made of laminated silicon steel and 8 pole surface-mounted-type PMs with Br = 1.28 T. The 12-slot stator has three-phase concentrated windings. The diameter of the coil is 1.2 mm, and the number of turns is 17. Between the high-speed rotor and the stator, 20 modulating ferromagnetic pole pieces made of laminated silicon steel are added. These pole pieces serve as the low-speed rotor and are used to modulate the space harmonics of the air-gap flux density.

![Fig. 1. Magnetic-geared machine](image-url)
2.2 Operational principle and gear ratio

The high-speed rotor is driven by the magnetomotive force due to the coil windings using the operational principle of conventional brushless motors. The low-speed rotor is rotated by employing the operational principle of an MG.

The torque transmission of the MGM is based on the modulation of a magnetic field using pole pieces between the stator and high-speed rotor. As shown in (1), the number of slots in the stator is \( N_s \), the number of magnetic pole pairs of the high-speed rotor is \( N_i \), and the number of pole pieces of the low-speed rotor is \( N_s \) [3-6].

\[
(2p-1)N_s = (2n-1)N_i \pm 2(2m-1)N_h \quad (1)
\]

where \( m \), \( n \), and \( p \) are positive integers. In addition, the gear ratio \( G_r \) of the MGM is defined by

\[
G_r = \mp \frac{(2n-1)N_i}{2(2m-1)N_h} \quad (2)
\]

In this paper, \( N_h = 4 \), \( N_i = 20 \), \( N_s = 12 \), \( m = n = p = 1 \) are selected. The gear ratio \( G_r \) is 2.5 by (2).

3. Reduction of Transmission Torque Ripple of MGM

3.1 Design of the high-speed rotor

The design of high-speed rotor was carried out before optimal design of the ferromagnetic pole pieces for transmission torque ripple reduction. The design of high-speed rotor was conducted without the ferromagnetic pole pieces. The torque of MGM is transmitted through the ferromagnetic pole pieces. Therefore, high-speed rotor design is more advantageous than the stator design for transmission torque ripple reduction. The design parameters of the high-speed rotor are the pole arc angle and thickness of PMs. The transmission torque ripple can be reduced by designing pole arc angle of the PMs, but by reducing the amount of the PMs which results in a reduced output. Therefore, pole arc angle of the PMs minimizing the transmission torque ripple was designed, and thickness of PMs was adjusted to satisfy the output power. Fig. 2 shows the rated torque and torque ripple according to the pole arc angle of the PMs. As shown in Fig. 2, it can be seen that the torque ripple was reduced the most when the pole arc angle of the PMs is 36°. However, rated torque was decreased to 0.28 Nm than when pole arc angle of the PMs is 45°. On this, thickness of PMs was designed to satisfy the output power. Fig. 3 shows the rated torque and torque ripple according to the thickness of PMs. As a result, 2.35 mm thickness of PMs, up from 0.35 mm initial model, satisfies the objective output power. After the insertion of the ferromagnetic pole pieces into derived model, analysis was proceeded to design the optimal ferromagnetic pole pieces.

Thus, transmission torque ripple is 65.81% and it is confirmed that the transmission torque ripple is increased considerably by ferromagnetic pole pieces. Therefore, optimal design of the ferromagnetic pole pieces to reduce the transmission torque ripple is proceeded in the next section.

3.2 Response surface methodology

Response surface methodology (RSM) is one of designs of experiment (DOE). The optimal condition of the RSM is determined through an ANOVA or the DOE. Then, an analysis is conducted of the interaction formula for input variables \( x_1, \cdots x_n \) and output value \( y \). In other words, the RSM is used to find the optimal response condition based on several factors. If there is a radius of curvature among the response variables, a response surface design is used to identify the relationship between two or more factors. When an unknown function between the design variables \( x_1, \cdots x_n \) and the dependent variable \( y \) is expressed as \( f(x) \). This unknown function is called a response function. When the number of design variables \( x \) is \( k \) and the response function \( y \) is assumed to be secondary regression model, \( y \) can be expressed as

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\[ y = \beta_0 + \sum_{i=1}^{k} \beta_i x_i + \sum_{i=1}^{k} \beta_{ij} x_i x_j + \sum_{i=1}^{k} \beta_{i} x_i + \varepsilon \]  

(3)

where \( \varepsilon \) represents a statistical error, which is generally assumed to be the normal distribution with a mean of zero and variance of \( \sigma^2 \) [7-14]. Therefore, the response function estimated from the approximation function is expressed in vector form as

\[ y = XE + \varepsilon \]  

(4)

where \( X \) is the matrix of the design variables, \( E \) is the regression coefficient vector, and \( \varepsilon \) is the random error vector. In addition, the regression coefficient vector is estimated using the least-squares method that takes the square sum of random errors as the minimum. The least-square estimator of the regression coefficient vector is [7-14]

\[ E = (X'X)^{-1}X'y. \]  

(5)

3.3 Box-Behnken design

The RSM is divided into a central composite design (CCD) and the BBD used in this study. As shown in Fig. 4, each experiment point of a design with three factors can be illustrated. The BBD is used to efficiently estimate the first and second terms of the response-surface-estimate equation, when it is certain that factors are not all low level or high level at the same time, or when it is certain that all experiments are conducted in a stable region. The number of experiments that use the BBD is relatively smaller than that of the CCD. BBD is useful when experiment costs are considerably expensive or practically impossible to conduct experiments at the factorial point.

On the other hand, CCD allows for an experiment that uses a factorial point and an axial point, while BBD allows for an experiment using the center point of the factorial point. When CCD is used, a design at the axial point is needed, as shown in Fig. 4(a).

For a design at the axial point, when there are three design variables, \( a \) has a value of ±1.682, which has an experiment point 1.682 times larger than the maximum value of the preset design variable and 1.682 times smaller than the minimum value [7-8]. In this case, the range of the preset design variable becomes larger, and the shape of the proposed low-speed rotor may not be formed, or design variables that cannot be designed are generated. This results in difficulties in determining the range of the initial design variables. Thus, in this paper, the shape design of the MGM was performed using BBD. The design variables for reduction of the transmission torque ripple of the MGM are shown in Table 1 and Fig. 5. The experiment, based on BBD, was set up as shown in Table 2.

3.4 Polynomial regression analysis

Polynomial regression analysis includes second or third order term for dependent variables, and interaction term

![Fig. 4. Response surface methodology: (a) Central composite design; (b) Box-Behnken design](image)

![Table 1. Design variables for reduction of transmission torque ripple](image)

<table>
<thead>
<tr>
<th>Level</th>
<th>Design variable</th>
<th>( \alpha ) (mm)</th>
<th>( \beta ) (deg)</th>
<th>( \gamma ) (mm)</th>
</tr>
</thead>
<tbody>
<tr>
<td>Low</td>
<td>-1</td>
<td>28</td>
<td>1</td>
<td>3</td>
</tr>
<tr>
<td>Central</td>
<td>0</td>
<td>29.5</td>
<td>2.5</td>
<td>5.5</td>
</tr>
<tr>
<td>High</td>
<td>+1</td>
<td>31</td>
<td>4</td>
<td>8</td>
</tr>
</tbody>
</table>

![Table 2. Box-Behnken design](image)

<table>
<thead>
<tr>
<th>Standard Order</th>
<th>Coded factor</th>
<th>Uncoded factor</th>
</tr>
</thead>
<tbody>
<tr>
<td>( \alpha )</td>
<td>( \beta )</td>
<td>( \gamma )</td>
</tr>
<tr>
<td>( x_1 )</td>
<td>( x_2 )</td>
<td>( x_3 )</td>
</tr>
<tr>
<td>1</td>
<td>-1</td>
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</tr>
<tr>
<td>2</td>
<td>+1</td>
<td>-1</td>
</tr>
<tr>
<td>3</td>
<td>-1</td>
<td>+1</td>
</tr>
<tr>
<td>4</td>
<td>+1</td>
<td>+1</td>
</tr>
<tr>
<td>5</td>
<td>-1</td>
<td>0</td>
</tr>
<tr>
<td>6</td>
<td>+1</td>
<td>0</td>
</tr>
<tr>
<td>7</td>
<td>-1</td>
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</tr>
<tr>
<td>8</td>
<td>+1</td>
<td>0</td>
</tr>
<tr>
<td>9</td>
<td>0</td>
<td>-1</td>
</tr>
<tr>
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<td>0</td>
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</tr>
<tr>
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<tr>
<td>12</td>
<td>0</td>
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<tr>
<td>13</td>
<td>0</td>
<td>0</td>
</tr>
</tbody>
</table>
between dependent variables. In order to design the experiment using the BBD and determine whether each factor is a significant term using the ANOVA, an f-test is used. In Tables 3 to 4, degree of freedom (DF) represents the number of points that can be freely changed under the given conditions; SSR is sum of square for regression; SSE is sum of square for residuals; and MS is mean square. F-value represents the test statistic for t-verification, which is used to verify whether the regression equation is significant in explaining the property of data. Where, is result of the experiment, is average of the values and is predicted value by the regression equation. In addition, p-value is the probability of significance.

When the p-value derived from this test is smaller than 0.05, the corresponding factor is considered to be significant. However, if the p-value is greater than 0.05, the corresponding factor is not significant. Accordingly, this factor is not considered to be one that affects dependent variables. Therefore, when performing an ANOVA, it is necessary to remove the factors of which p-value are greater than 0.05 (i.e., the factors that are not significant) sequentially from larger values [7-14].

Through this process, factors of the terms of interaction and factors of the terms of squares whose p-values are greater than 0.05 were removed. Analysis was performed according to the BBD, and the results were used to perform a regression analysis of design variables , , and of the low-speed rotor. As a result of the variance analysis of the transmission torque, all square terms and interaction terms were found to be insignificant and were removed. For the transmission torque ripple, of the square term and of the interaction term were found to be insignificant and were removed. The ANOVA results of the transmission torque and the ripple are shown in Table 3 and 4, respectively.

In Table 3, and in the linear term have a p-value of 0.05 or greater and thus, they are not significant. However, because the linear term itself is significant, it is acceptable to perform regression analysis without removing these terms.

Thus, the transmission torque ripple can be estimated with a secondary model. However, for the transmission torque, only the main effect works, and therefore, the transmission torque can be estimated with a linear model. Regression analysis equations for each design variable are as follows:

\[ f_T(a, \beta, \gamma) = 6.12 - 0.0443a - 0.163\beta + 0.09\gamma \]  
\[ f_{TR}(a, \beta, \gamma) = 344.2 - 6.66a - 176.3\beta + 10.24\gamma + 15.865\beta\gamma - 0.987\gamma^2 + 3.143a\beta \]

where \( f_T(a, \beta, \gamma) \) and \( f_{TR}(a, \beta, \gamma) \) are the estimation functions of the transmission torque and the transmission torque ripple, respectively.

### 3.5 Design results and numerical analysis

Considering the transmission torque and transmission torque ripple characteristic of the MGM, the multiobjective design optimization model of a low-speed model can be defined as

\[ \text{Maximize} : f_T(a, \beta, \gamma) \]  
\[ \text{Minimize} : f_{TR}(a, \beta, \gamma) \]

The overlaid contour plot for design variables of the proposed low-speed rotor shape is shown in Fig. 6. In
general, the overlaid contour plot has one design variable on the x-axis and the other on y-axis. If there are three or more variables, design variable should be fixed to a certain level. In the overlaid contour plot, the contour plot for each response is overlaid on one graph, and each contour plot represents the boundary of the response function value defined in the legend. The contour plot drawn with a solid line represents the lower limit, and the contour plot drawn with a dotted line represents the upper limit. The white region is the optimal region that satisfies all ranges indicated in the legend. Therefore, through an analysis of the response surface and the overlaid contour plot, the optimal design variable \( \alpha \) for the proposed low-speed rotor was determined to be 28 mm. In addition, \( \beta \) was 2.4°, and \( \gamma \) was 8 mm. Fig. 7 shows the magnetic flux density of the initial and optimized models. Fig. 8 shows the transmission torque waveforms of the initial and optimized models. As shown in Fig. 8(a), after optimization, the transmission torque ripple was reduced by 130.8% from that of the initial model. In addition, as shown in Fig. 8(b), the 25th, 37th, 49th harmonic components are dominant. In fact, the 25th harmonic component is the main cause of for higher torque ripples in the MGM. Thus, it is mandatory to discard the specific 25th torque harmonic component as much as possible. Remarkably, it is shown that the proposed design method is very effective in compensating for the targeted 25th torque harmonic component.

4. Conclusion

In this paper, an effective shape to reduce transmission torque ripple of the MGM is derived from a BBD. An analysis is conducted between independent variables selected by BBD and reaction variables using the non-linear FEM based on 2-D numerical analysis. In addition, the regression equation of the reaction variables is derived according to the independent variable by multiple regression analysis and an ANOVA. The validity of the optimized design is demonstrated by comparing characteristics of the optimized model (derived from response surface analysis) and the initial model. As a result, the transmission torque ripple was reduced by 130.8% from that of the initial model.

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